

IN THE CLAIMS

Please amend the claims as follows:

1. (Cancelled)
2. (Previously Presented) Sealing arrangement according to claim 10 wherein in an unpressurized state, an inner radius of the U-cup seal decreases continuously from the low-pressure side towards the inner sealing lip.
3. (Previously Presented) Sealing arrangement according to claim 2, wherein in the unpressurized state, the inner radius of the U-cup seal conically decreases from the low-pressure side towards the inner sealing.
4. (Previously Presented) Sealing arrangement according to claim 10 wherein in an unpressurized state, the abutment surface of the U-cup seal is, in part, convex.
5. (Previously Presented) Sealing arrangement according to claim 10 wherein the U-cup has an outer concave surface adjacent a groove bottom in the unpressurized state.
6. (Previously Presented) Sealing arrangement according to claim 10 wherein in an unpressurized state, an outer radius of the U-cup seal increases continuously from the low-pressure side towards the outer sealing lip.
7. (Previously Presented) Sealing arrangement according to claim 10 wherein the U-cup seal has an outer

surface facing away from the movable piston and that in an unpressurized state, an outer edge of the U-cup seal is formed convex in a transition region between abutment surface and outer surface.

8. (Previously Presented) Sealing arrangement according to claim 10 wherein the U-cup seal has an inner surface facing the movable machine part (1), with calotte shells as a microstructure.

9. (Cancelled)

10. (Currently Amended) Sealing arrangement comprising a U-cup of a viscoplastic synthetic material, a stationary machine part, and a translatory movable machine part with an outer radius  $R$ , wherein the U-cup is disposed as a contacting joint under radial prestress between the stationary machine part and the movable machine part in a profiled section of the stationary machine part, wherein the U-cup has a radially outer and a radially inner sealing lip on a high-pressure side, wherein the stationary and the movable machine parts are separated on a low-pressure side by a sealing gap width  $B$ , wherein an abutment surface of the U-cup abuts a radially oriented region of the profiled section on the low-pressure side, wherein the U-cup has an inner radius and an outer radius, wherein both in the unpressurized state and in the pressurized state, the inner radius of the U-cup in the region of the abutment surface is larger than the sum of  $R$  and  $B$  and wherein the U-cup comprises an inner surface facing the movable machine part, wherein the inner surface comprises several liquid dragging bore reliefs formed as recesses in the inner surface of the

U-cup for transferring hydraulic liquid, disposed on a surface of the translatable movable machine part, from a low pressure region to a high pressure region upon translatable movement of the movable machine part, wherein the recesses each extend in an axial direction from the low pressure side N of the U-cup towards the inner sealing lip, and the radial depth of the individual recesses decreases from the low-pressure side N of the U-cup towards the inner sealing lip, the recesses, as well as an inner surface region with the recesses, being spaced apart from the moveable machine part and the sealing lip in an unpressurized state.